# **MIET2510**

**Mechanical Design** 

#### Week 3 – Kinematic Analysis of Mechanism – Part 3

School of Science and Technology, RMIT Vietnam





- 1. Acceleration Analysis
- 2. Analytical Solution for Acceleration Analysis
- 3. Homework and Further Discussion Topics



# **1. Acceleration Analysis**

- After velocity analysis the next step is to determine the acceleration of all links and points of interest in the mechanism or machine. We need to know the accelerations to calculate the dynamic force.
- The dynamic force will contribute to the stresses in the links and other components.
- We will focus on the analytical solution for acceleration analysis.



### **1. Acceleration Analysis**

• Angular Acceleration will be defined as  $\alpha$  and linear acceleration as A.



$$\mathbf{R}_{PA} = p e^{j\theta}$$

$$\mathbf{V}_{PA} = \frac{d\mathbf{R}_{PA}}{dt} = p j e^{j\theta} \frac{d\theta}{dt} = p \omega j e^{j\theta}$$
$$\mathbf{A}_{PA} = \frac{d\mathbf{V}_{PA}}{dt} = \frac{d\left(p \omega j e^{j\theta}\right)}{dt}$$
$$\mathbf{A}_{PA} = j p \left(e^{j\theta} \frac{d\omega}{dt} + \omega j e^{j\theta} \frac{d\theta}{dt}\right)$$
$$\mathbf{A}_{PA} = p \alpha j e^{j\theta} - p \omega^2 e^{j\theta}$$
$$\mathbf{A}_{PA} = \mathbf{A}_{PA}^t + \mathbf{A}_{PA}^n$$

 $\mathbf{A}_{PA} = p\alpha \left(-\sin\theta + j\cos\theta\right) - p\omega^2 \left(\cos\theta + j\sin\theta\right)$ 



#### **Conventional Pin-Jointed Four-bar linkage**



• From previous part (velocity analysis), we have:

$$ja\omega_2 e^{j\theta_2} + jb\omega_3 e^{j\theta_3} - jc\omega_4 e^{j\theta_4} = 0$$

 If we take derivative from velocity equation for the loop in 4-link mechanism, then:

$$\frac{\mathrm{d}}{\mathrm{dt}} \left( ja\omega_2 e^{j\theta_2} + jb\omega_3 e^{j\theta_3} - jc\omega_4 e^{j\theta_4} \right) = 0$$

$$(j^2 a \omega_2^2 e^{j\theta_2} + ja\alpha_2 e^{j\theta_2}) + (j^2 b \omega_3^2 e^{j\theta_3} + jb\alpha_3 e^{j\theta_3})$$
$$- (j^2 c \omega_4^2 e^{j\theta_4} + jc\alpha_4 e^{j\theta_4}) = 0$$



• Simplifying the grouping terms:

$$\left(a\alpha_2 je^{j\theta_2} - a\omega_2^2 e^{j\theta_2}\right) + \left(b\alpha_3 je^{j\theta_3} - b\omega_3^2 e^{j\theta_3}\right) - \left(c\alpha_4 je^{j\theta_4} - c\omega_4^2 e^{j\theta_4}\right) = 0$$

• Above equation is in fact:

$$\mathbf{A}_{A} + \mathbf{A}_{BA} - \mathbf{A}_{B} = 0$$
$$\mathbf{A}_{A} = \left(\mathbf{A}_{A}^{t} + \mathbf{A}_{A}^{n}\right) = \left(a\alpha_{2} je^{j\theta_{2}} - a\omega_{2}^{2} e^{j\theta_{2}}\right)$$
$$\mathbf{A}_{BA} = \left(\mathbf{A}_{BA}^{t} + \mathbf{A}_{BA}^{n}\right) = \left(b\alpha_{3} je^{j\theta_{3}} - b\omega_{3}^{2} e^{j\theta_{3}}\right)$$
$$\mathbf{A}_{B} = \left(\mathbf{A}_{B}^{t} + \mathbf{A}_{B}^{n}\right) = \left(c\alpha_{4} je^{j\theta_{4}} - c\omega_{4}^{2} e^{j\theta_{4}}\right)$$



 Our problem is to find angular acceleration α<sub>3</sub> and α<sub>4</sub> knowing the angular input acceleration α<sub>2</sub>, the link length, all link angles, and angular velocities.

$$\alpha_3 = f(a, b, c, d, \theta_2, \theta_3, \theta_4, \omega_2, \omega_3, \omega_4, \alpha_2)$$
  
$$\alpha_4 = g(a, b, c, d, \theta_2, \theta_3, \theta_4, \omega_2, \omega_3, \omega_4, \alpha_2)$$



 The strategy of solution will be the same as was done for position and velocity analysis.

$$\begin{bmatrix} a\alpha_2 \ j(\cos\theta_2 + j\sin\theta_2) - a\omega_2^2 (\cos\theta_2 + j\sin\theta_2) \end{bmatrix} \\ + \begin{bmatrix} b\alpha_3 \ j(\cos\theta_3 + j\sin\theta_3) - b\omega_3^2 (\cos\theta_3 + j\sin\theta_3) \end{bmatrix} \\ - \begin{bmatrix} c\alpha_4 \ j(\cos\theta_4 + j\sin\theta_4) - c\omega_4^2 (\cos\theta_4 + j\sin\theta_4) \end{bmatrix} = 0$$



• The real component is

 $-a\alpha_2\sin\theta_2 - a\omega_2^2\cos\theta_2 - b\alpha_3\sin\theta_3 - b\omega_3^2\cos\theta_3 + c\alpha_4\sin\theta_4 + c\omega_4^2\cos\theta_4 = 0$ [1]

• The imaginary component is:

 $a\alpha_2\cos\theta_2 - a\omega_2^2\sin\theta_2 + b\alpha_3\cos\theta_3 - b\omega_3^2\sin\theta_3 - c\alpha_4\cos\theta_4 + c\omega_4^2\sin\theta_4 = 0$  [2]



• Solving [1] and [2] simultaneously we get

$$\begin{aligned} \alpha_{3} &= \frac{(a\alpha_{2}\cos\theta_{2} - a\omega_{2}^{2}\sin\theta_{2} - b\omega_{3}^{2}\sin\theta_{3} + c\omega_{4}^{2}\sin\theta_{4})(c\cos\theta_{4})}{(c\sin\theta_{4})(b\cos\theta_{3}) - (b\sin\theta_{3})(c\cos\theta_{4})} \\ &- \frac{(c\sin\theta_{4})(a\alpha_{2}\sin\theta_{2} + a\omega_{2}^{2}\cos\theta_{2} + b\omega_{3}^{2}\cos\theta_{3} - c\omega_{4}^{2}\cos\theta_{4})}{(c\sin\theta_{4})(b\cos\theta_{3}) - (b\sin\theta_{3})(c\cos\theta_{4})} \\ \alpha_{4} &= \frac{(a\alpha_{2}\sin\theta_{2} + a\omega_{2}^{2}\cos\theta_{2} + b\omega_{3}^{2}\cos\theta_{3} - c\omega_{4}^{2}\cos\theta_{4})(b\cos\theta_{3})}{(c\sin\theta_{4})(b\cos\theta_{3}) - (b\sin\theta_{3})(c\cos\theta_{4})} \\ &- \frac{(b\sin\theta_{3})(a\alpha_{2}\cos\theta_{2} - a\omega_{2}^{2}\sin\theta_{2} - b\omega_{3}^{2}\sin\theta_{3} + c\omega_{4}^{2}\sin\theta_{4})}{(c\sin\theta_{4})(b\cos\theta_{3}) - (b\sin\theta_{3})(c\cos\theta_{4})} \end{aligned}$$



• Having obtained  $\alpha_3$  and  $\alpha_4$  we can obtain the accelerations.

$$egin{aligned} \mathbf{A}_A &= a lpha_2 \left( -\sin heta_2 + j \cos heta_2 
ight) - a \omega_2^2 \left( \cos heta_2 + j \sin heta_2 
ight) \ \mathbf{A}_{BA} &= b lpha_3 \left( -\sin heta_3 + j \cos heta_3 
ight) - b \omega_3^2 \left( \cos heta_3 + j \sin heta_3 
ight) \ \mathbf{A}_B &= c lpha_4 \left( -\sin heta_4 + j \cos heta_4 
ight) - c \omega_4^2 \left( \cos heta_4 + j \sin heta_4 
ight) \end{aligned}$$



# **3. Homework and Further Topics**

- Graphical Approach in Acceleration Analysis.
- Sample Problems in the Reading Notes.
- Acceleration of Any Point in the Linkage.



Thank you for your attendance :D



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• Design of Machinery by Robert L. Norton.

