# **MIET2510**

**Mechanical Design** 

#### Week 8 – Fasteners and Connections – Part 2

School of Science and Technology, RMIT Vietnam



# Welded, Brazed, and Soldered Joints

- A weld is a **permanent joint** that results from diffusion of material across an interface and may include a filler metal. Welding applied to components with similar materials which have same melting temperature.
- Brazing and soldering develop an interface from the adhesion between members and a filler metal. Brazing occurs above 450 C.





# Welded Joints compared to Fasteners

- Advantages;
  - Inexpensive
  - No danger of joint loosening
- Disadvantages;
  - Produce residual stresses
  - Metallurgical changes occur
  - Disassembly is a problem
  - Weld quality depends on operator skill



# Welding methods

- Fusion Welding involves melting and coalescing materials by means of heat, usually supplied by chemical, electrical, or high energy means. Ex: Oxygen gas welding.
- <u>Solid-state welding</u> is a group of welding processes that produces coalescence at temperatures essentially below the melting point of the base materials being joined, without the addition of brazing filler metal

https://bamason2.github.io/miet2510-module/notes/fasteners\_and\_connectors.html



# **Welding Methods**





# **Configurations of welds – Spot Welds**

- <u>Spot welding</u> is one of the most commonly used processes in sheet metal fabrication and automotive body panel assembly.
- In spot welding, the tips of two opposing solid, cylindrical electrodes touch a lap joint of two sheet metals, and resistance heating produces a spot weld. To obtain a strong bond in the weld, pressure is applied until the current is turned off and the weld has solidified.





## **Configurations of welds – Line Welds**

- The line welding processes result in weld beads or lines that have a small width compared to their length.
- Several different line joints can be achieved, and **fillet weld** is of the most



common ones.

Type of weld								
Bead	Fillet	Plug	Groove					
Deau		slot	Square	V	Bevel	U	J	
				$\checkmark$	$\bigvee$	$\bigtriangledown$	V	



#### **Configurations of welds – Line Welds**

• In Parallel and Transverse Loading, shear stress is:

$$\tau = \frac{P}{t_e L_w} = \frac{P}{0.707 h_e L_w} = \frac{1.414P}{h_e L_w},$$

where  $t_e$  is the weld throat length,  $h_e$  is the weld leg length, and  $L_w$  is the weld length. Thus, to avoid failure, the following should be valid:

$$\frac{S_{sy}}{n_s} = \frac{P}{t_e L_w}$$





# **Configurations of welds – Line Welds**

• In Torsional Loading, the shear stress is;

$$\tau_t = \frac{Tr}{J},$$

where r is the distance from the centroid of the weld group to farthest point in the weld, T is the applied torque, and Jis the polar area moment of inertia. The critical section for torsional loading is the throat section, as it is for parallel and transverse loading.

$$J = t_e J_u = 0.707 h_e J_u.$$



# **Tensile Properties of Fillet Welds**

Weld	Throat Area	Location of G	Unit Second Polar Moment of Area
1. $G$ $d$ $d$ $y$	A = 0.707 hd	$\overline{x} = 0$ $\overline{y} = d/2$	$J_u = d^3/12$
2. $( + b \rightarrow )$ y $\overline{y}$ $\overline{x}$ $\overline{x}$ $\overline{x}$	A = 1.414hd	$\overline{x} = b/2$ $\overline{y} = d/2$	$J_{\mu} = \frac{d(3b^2 + d^2)}{6}$
3. $b \longrightarrow b$ $\overline{y}$ $G$ $d$ $d$ $\overline{y}$ $\overline{x}$ $+$	A = 0.707h(b + d)	$\overline{x} = \frac{b^2}{2(b+d)}$ $\overline{y} = \frac{d^2}{2(b+d)}$	$J_{u} = \frac{(b+d)^{4} - 6b^{2}d^{2}}{12(b+d)}$
4. $(\leftarrow b \rightarrow)$ $\overline{y}$ $G$ $d$ $\rightarrow   \overline{x} $ $\leftarrow$	A = 0.707h(2b + d)	$\overline{x} = \frac{b^2}{2b+d}$ $\overline{y} = d/2$	$J_{\mu} = \frac{8b^3 + 6bd^2 + d^3}{12} - \frac{b^4}{2b + d}$
5. $\leftarrow b \rightarrow$	A = 1.414h(b + d)	$\overline{x} = b/2$ $\overline{y} = d/2$	$J_u = \frac{(b+d)^3}{6}$
6.	$A = 1.414\pi hr$		$J_u = 2\pi r^3$

\*G is the centroid of weld group; h is weld size; plane of torque couple is in the plane of the paper; all welds are of unit width.

In bending, the welded joint experiences a transverse shear stress as well as a normal stress.

In bending, the normal stress is:

$$\sigma = \frac{Mc}{I}$$

$$= t_e I_u = 0.707 h_e I_u$$

where c is the distance from the neutral axis to the outer fiber.

In bending, the shear stress is similar to the shear stress in transverse loading.



#### Configurations of welds – Area Welds

- Many welding operations produce a weld joint over an area, such as cold rolling, diffusion bonding, or friction welding; this is also the situation for brazed and soldered joints.
- From a design analysis standpoint, these joints can be treated like adhesively bonded joints as discussed in next section, but the allowable stresses over the interfaces are accordingly larger.



Figure 16.29: Examples of adhesively bonded joints. (a) Butt; (b) scarf; (c) lap; (d) bevel; (e) double lap; (f) increased thickness; (g) strap.



Adhesive bonding is the process of joining materials chemically through the formation of interatomic or intermolecular bonds. Adhesive bonding can be used to join a wide variety of materials (metal to metal, metal to ceramic, metal to polymer, etc.) for both structural and non-structural uses.

	Epoxy	Polyurethane	Modified acrylic	Cyanoacrylate	Anaerobic		
Impact resistance	Poor	Excellent	Good	Poor	Fair		
Tension-shear strength, MPa	15-22	12-20	20-30	18.9	17.5		
Peel strength, $N/m^{a}$	< 523	14,000	5250	< 525	1750		
Service temperature range, °C	-55-120	-40-90	-70-120	-55-80	-55-150		
Solvent resistance	Excellent	Good	Good	Good	Excellent		
Moisture resistance	Good-excellent	Fair	Good	Poor	Good		
Odor	Mild	Mild	Strong	Moderate	Mild		
Toxicity	Moderate	Moderate	Moderate	Low	Low		
Flammability	Low	Low	High	Low	Low		
<sup>a</sup> Peel strength varies widely depending on surface preparation and quality.							



#### **Adhesive Bonding – Common Joints**





# **Adhesive Bonding - Advantages**

- Uniform stress distribution with resultant increased life
- Reduced weight
- Improved fatigue resistance
- Ability to join thick or thin materials
- Ability to join dissimilar materials
- The adhesive results in leakproof joints
- Vibration-damping and insulation properties
- Economic advantages associated with cost of adhesive and ease of assembly



#### **Adhesive Bonding - Disadvantages**

- Possible need for extensive surface preparation
- Service temperatures are more limited compared to welds, brazes, or solders
- Tendency to creep under sustained load
- Questionable long-term durability



## Adhesive Bonding – Stress Analysis

• For typical lap joint configuration:

$$\tau_{\rm max} = 2\tau_{\rm avg} = \frac{2P}{bL}$$

• For scarf joint under axial load, normal stress and shear stress is

$$\sigma_n = \sigma_x \sin \theta = \frac{P}{bt_m} \sin^2 \theta, \qquad \tau = \sigma_x \cos \theta = \frac{P}{bt_m} \sin \theta \cos \theta = \frac{P}{2bt_m} \sin 2\theta.$$

• For scarf joint under bending, the shear stress is

$$\tau = \frac{\sigma_n}{\tan \theta} = \frac{6M}{bt_m^2} \sin^2 \theta \frac{\cos \theta}{\sin \theta} = \frac{6M}{bt_m^2} \sin \theta \cos \theta = \frac{3M}{bt_m^2} \sin 2\theta.$$

• For scarf joint under torsional loading, the shear stress is

$$\tau = \frac{2Tr\sin\theta}{\pi \left(r_o^4 - r_i^4\right)}. \quad \sigma_n = 0.$$



Thank you for your attendance :D



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- Mechanical Design of Machine Components (2nd) by Ansel C.Ugural.
- Mechanical Engineering Design (10th) by Richard G.Budynas and J. Keith Nisbett.
- Theory of Machines and Mechanisms (5th) by John J.Uicker, Gordon R.Pennock, Joseph E. Singley.

